

# Boosting Wind Harvest: FOPID Pitch Angle Controller for Turbines

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**Abstract**—Wind turbine blades are subjected to a variety of loads, including aerodynamic and gravitational loads. These loads produce aerodynamic strain and vibration in the blades, resulting in rotor blade damage and a reduction in the wind turbine's system efficiency. This could be prevented by implementing a proper pitch angle controller that plays a crucial role in boosting the energy capture and overall performance of wind power systems. The conventional Proportional-Integral-Derivative, *PID*, controller has been widely utilized for pitch control, but it often faces challenges in meeting the requirements of complex and dynamic wind conditions. To address these limitations, this study explores the implementation of the Fractional Order Proportional-Integral-Derivative, *FOPID*, controller for wind turbine pitch control. This paper presents a comparative analysis between the *PID* and *FOPID* controllers for wind turbine pitch control. The performance of both controllers is evaluated through Simulink. The results demonstrate that the *FOPID* controller exhibits superior performance in terms of faster response time, and improved steady-state error compared to the *PID* controller.

**Index Terms**—*PID* Controller, *FOPID* Controllers, Wind Turbine, Pitch Controller

## I. INTRODUCTION

Renewable energy is derived from natural processes that can provide sustainable, non-ending, energy with minimal impact on the environment. When fossil fuels are used to create energy, they emit dangerous greenhouse gases like carbon dioxide. To address the climate catastrophe, a shift from fossil fuels to renewable energy is essential [1]. There are multiple sources that produce clean energy such as solar, wind, water, heat, and nuclear. However, they differ in their location, equipment, and efficiency. Many studies proved that relying on sustainable energy sources decreases air pollution by lowering the reliance on burning fuels, emitting no greenhouse gases, diversifying the energy supply, and promoting economic growth.

Several utility organizations are investing in researching renewable energy sources that will be implemented as power generators. Some of these organizations have concentrated their research on specific sources of alternative energy, such as fuel cells, solar panels, wind turbines (*WT*), and micro-turbines. The main distinguishing feature of *WT*, in comparison to other power production systems, is that the power input rate cannot be regulated. However, wind speed and power demand in *WT* systems change based on numerous factors such as climatic conditions and time. *WT* systems deliver substantial obstacles due to their undefined parameters and unpredictable environmental disturbances. In addition, under different wind speed conditions, the *WT* system is subject to the aerodynamic, gravitational, centrifugal, gyroscopic, and operational loads. Nonetheless, these drawbacks can be overcome by adopting a pitch control system that enhances the *WT*'s performance. This control strategy starts by utilizing a controller to check the turbine's power output multiple times per second. The rotor blades adjust the angle of attack by turning slightly out of the wind in response to a signal sent to the blade pitch mechanism if the wind speed exceeds the operational limit. These blades are reoriented toward the wind once it subsides.

The rotational speed of a *WT* refers to the speed at which the rotor blades are spinning. It is an essential factor in determining the performance and power output of the turbine. This rotational speed depends on various factors, including the design of the turbine, wind speed, and electrical grid requirements. *WTs* are designed to operate within a specific range of rotational speeds to maximize energy production while maintaining safe operating conditions. The rotational speed is influenced by the wind speed and the aerodynamic properties of the rotor blades. As the wind speed increases, the rotor blades experience greater aerodynamic forces, causing them to rotate at a faster speed. *WTs* are equipped with control

systems that adjust the pitch angle of the blades to optimize power production and maintain a desired rotational speed. It's important to note that *WTs* have a cut-in, and a cut-out speeds. The cut-in speed is the minimum wind speed at which the turbine starts generating power, while the cut-out speed is the maximum wind speed at which the turbine is shut down for safety reasons. The rotational speed is controlled within the operational range between the cut-in and cut-out speeds. Monitoring and controlling the rotational speed of *WTs* is an integral part of their operation and is typically managed by the turbine's control system, which constantly adjusts the pitch angle and other parameters to optimize performance and ensure safe operation.

There is a diversity of control strategies including *PID*, Model Predictive Control, fuzzy logic, and adaptive control. Each of them can be employed based on the specific requirements and complexity of the system. One common closed-loop feedback method in commercial control systems is the *PID* controller. The difference between the desired set point and the measured process variable is used to determine an error value. By adjusting variables, the controller tunes the process in an attempt to reduce mistakes. Its three parameters are tweaked to guarantee that the system will stay stable. *PID* controllers are used in conjunction with *WT* to achieve the desired pitch angle [2]. *FOPID* control has been applied in various fields, including robotics, chemical processes, power systems, and control systems with non-linear or time-delayed dynamics [3].

## II. LITERATURE REVIEW

*WT* pitch angle control has been a subject of extensive research as documented in several scholarly papers. In [4], the authors investigated the performance of *PI* and fractional *PI* (*FPI*) controllers on *WT* pitch angle. Their results showed enhanced performance, greater reliability, and improved generation efficiency of *FPI* in comparison to *PI*. In [5], a fuzzy logic pitch angle controller was introduced as an alternative to the *PI* controller with a better performance. In [6], the authors suggested the use of the *FOPID* controller in improving the *WT's* performance in conjunction with the Particle Swarm Optimization *PSO* to find the controller gains. In [7], the authors investigated the performance of *FOPID* controller for pitch control in a virtual 20 MW *WT* system, and by using "FAST" software. Their outcomes showed improved performance.

This study investigates the effectiveness of *FOPID* controller in designing a *WT* pitch regulator. In addition, it provides a comparison with the *PID* controller in terms of the overshoot, rise time, and settling time. The paper is organized as follows. The structure, and operation of the *WT* are explained in Section II, while the methodology including the *WT* model, pitch control system using *PID* and *FOPID* are given in Section III. The system simulation results are addressed in Section IV. Finally, conclusions are given in Section V.

## III. WIND TURBINE STRUCTURE AND OPERATION

A *WT* is a device that converts the kinetic energy of wind into electrical energy. It typically consists of a tower, rotor blades, a nacelle, and a generator. The tower provides height to capture the stronger and more consistent wind at higher altitudes. The rotor blades, usually three in number, are mounted on a hub and rotate when exposed to the force of the wind. The nacelle sits atop the tower and houses the gearbox, generator, and control systems. The drive train consists of two main components: the gearbox and the generator. The gearbox is used to increase the speed of the low-speed shaft connected to the blades to match the speed of the high-speed shaft connected to the generator. The generator is used to convert the mechanical energy of the high-speed shaft into electrical energy that can be transmitted to the grid, Fig. 1.

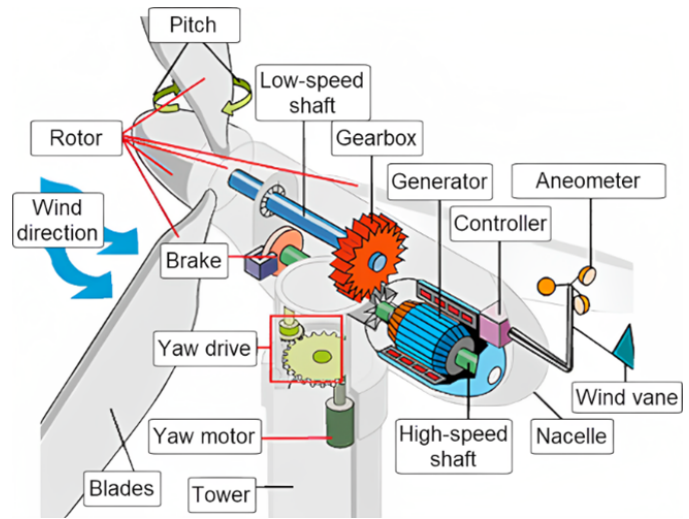


Fig. 1. The internal structure of a wind turbine and its components [8].

*WTs* are classified into two main types: horizontal-axis *WTs* (*HAWTs*) and vertical-axis *WTs* (*VAWTs*). The (*HAWTs*) have the rotor blades mounted on a horizontal axis, facing into the wind. The (*VAWTs*), on the other hand, have their rotor blades mounted on a vertical axis and can capture wind from any direction.

The operation of a *WT* is as follows. *WTs* are equipped with anemometers and wind vanes to measure wind speed and direction. These sensors provide crucial information for the turbine's operation and control systems. When the wind speed reaches a certain threshold, the *WT's* control system initiates the start-up sequence. The rotor blades begin to rotate slowly, driven by the wind's force. As the wind speed increases, the rotor blades rotate at a faster speed. The kinetic energy of the wind is transferred to the rotor, causing it to spin. The rotor is connected to a gearbox, which increases the rotational speed to match the optimal speed for the generator. The high-speed rotor shaft is connected to a generator within the nacelle. The generator converts the mechanical energy from the rotor into electrical energy. This electrical energy is typically in the form of alternating current *AC*. The generated *AC*

power from the generator is sent through power conditioning systems, such as power converters, to adjust the voltage and frequency to match the requirements of the electrical grid. This ensures compatibility with the existing power infrastructure. The conditioned electrical power is then transmitted through power lines to the electrical grid for distribution to consumers. *WTs* in wind farms are interconnected to create a collective power generation system, contributing to the overall electricity supply. *WTs* are equipped with sophisticated control systems that continuously monitor and optimize their performance. These systems adjust the pitch angle of the rotor blades to optimize power production, maintain safe operating conditions, and protect the turbine from extreme wind conditions. In cases of high wind speeds, severe weather conditions, or maintenance requirements, *WTs* can be automatically or manually shut down. The rotor blades are feathered (turned parallel to the wind) to minimize their exposure to strong winds, ensuring the safety and integrity of the turbine [9]. The operation of *WTs* is highly automated, with advanced control systems and monitoring technologies optimizing performance, maximizing energy production, and ensuring the safe and efficient operation of the turbines.

#### IV. METHODOLOGY

This study starts by an appropriate *WT* model, and followed by the *PID* and *FOPID* pitch control system. The Simulink is utilized for the design and analysis of the two types of control systems.

##### A. Wind turbine model

There are three terms in any *WT*, which are the cut-in, cut-out, and rated speed values. The cut-in speed is at which *WT* starts running. Below this wind speed, the wind energy is not sufficient to overcome the inertia of the rotor and thus there is no generated power. At high wind speed, the turbine is programmed to stop to avoid damaging the turbine or its surroundings. The cut-out wind speed is the stop wind speed. The rated wind speed is at which the rated power is achieved. Fig. 2 shows the output power of a 400W *WT*. It shows that the turbine's output power drops sharply at lower wind speeds. In addition, it shows that the available wind power increases eight times for every doubling of wind speed and decreases eight times for every halving of the wind speed.

This study focuses on modeling two parts of the *WT* that affect the pitch angle which are the actuator and the drive train. The actuator model describes a dynamic behavior between the measured pitch angle  $\beta$ , and the desired value  $\beta_d$ . The drive-train model describes the conversion of the rotational motion of the blades (caused in proportion to the torque,  $T$ , in  $N.m$ ) into electrical power (W in Watt). The proposed pitch control system is depicted in Fig. 3.

In [11], the authors derived a mathematical model of the actuator and the drive train of a 1MW *WT*, given in (1).

$$\frac{\beta}{\beta_d} = \frac{1}{0.5s + 1}; \quad \frac{W}{T} = \frac{0.5}{0.375s + 1} \quad (1)$$

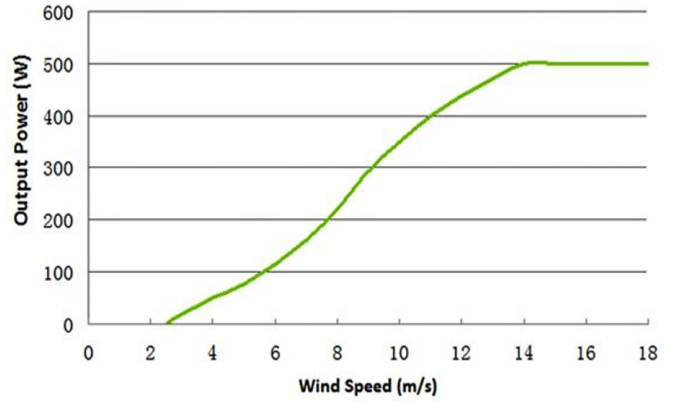


Fig. 2. The output power of X-400 *WT* [10].

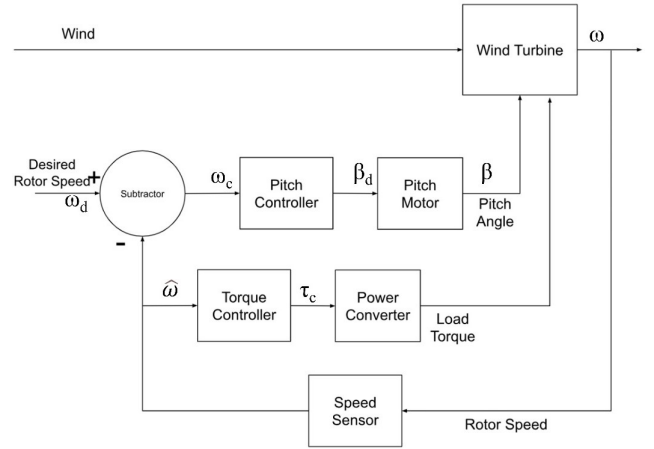


Fig. 3. Block diagram of the proposed pitch angle control system.

##### B. Pitch control systems

This study investigate two type of controllers which are the *PID* and the *FOPID*.

1) *PID controller*: Researchers have a variety of options when working with *PID* controller, including parallel, series, and the ideal configurations [12]. The parallel *PID* controller is expressed in (2).

$$G_{PID} = \frac{C(s)}{E(s)} = K_p + \frac{K_i}{s} + K_d \times s \quad (2)$$

$$c(t) = K_p e(t) + k_i \int_0^t e(t) dt + k_d \frac{de(t)}{dt}$$

where  $c(t)$  is the controller output at time  $t$ ,  $e(t)$  is the error signal at time  $t$ ,  $K_p$ ,  $K_i$ , and  $K_d$  are the proportional, integral, and derivative coefficients, respectively.

This study utilizes the three components ( $K_p$ ,  $K_i$ , and  $K_d$ ) of the ideal *PID* controller to adjust the pitch angle based on the difference between the desired angle and the actual angle measured by sensors. The control signal generated by the *PID* controller is used to adjust the pitch angle of the *WT* blades. The gains of the *PID* controller need to be tuned to achieve optimal performance. This tuning

process involves adjusting the gains experimentally to ensure stable and responsive control without excessive overshooting or oscillations. Details about the *PID* controller and its tuning with applications are discussed in [13], and [14]. Fig. 4 shows the structure of the *PID* controller.

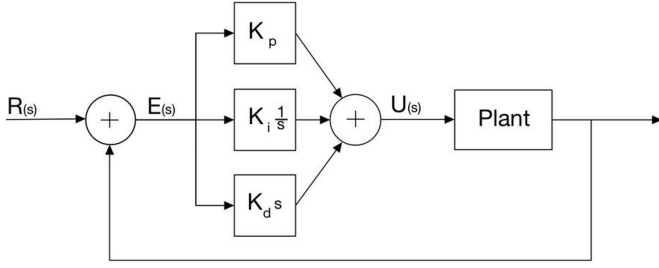


Fig. 4. Block diagram of *PID* controller.

2) *FOPID* controller: This controller is an extension of the traditional *PID* control. It introduces fractional calculus concepts to enhance the control performance [15]. Fig. 5 depicts the block diagram of the *FOPID* controller. Its transfer function is in (3).

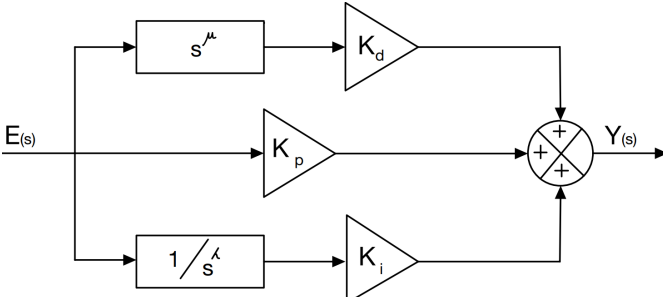


Fig. 5. Block diagram of *FOPID* controller.

$$TF(s) = K_p + K_d \times s^\mu + K_i \times \frac{1}{s^\lambda} \quad (3)$$

### C. Step performance

This study utilized the Simulink for the design and analysis of the *WT* pitch control system. The Matlab toolboxes: control system, optimization, and Fractional-order modeling and control of dynamic systems, *FOMCON* are used. Fig. 6, 7, 8 display the Simulink design of the *WT* pitch control system without a controller, with a tuned *PID* controller, and with a tuned *FOPID* controller.

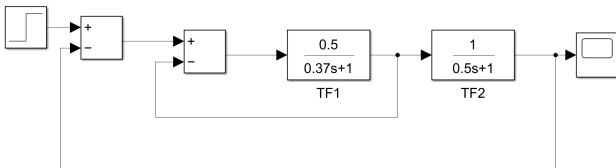


Fig. 6. Simulink diagram of uncontrolled wind turbine system.

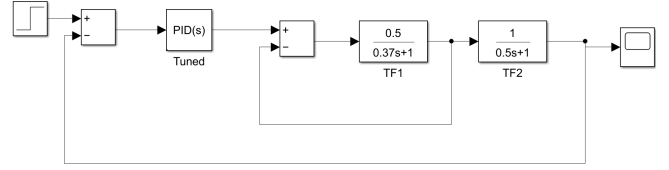


Fig. 7. Simulink of *WT* system controlled by a tuned *PID*.

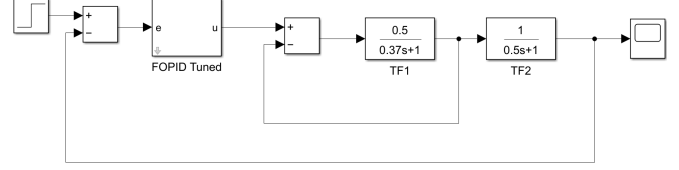


Fig. 8. Simulink of *WT* system controlled by a tuned *FOPID*.

## V. RESULTS

The Simulink tuner is used to adjust the gain parameters of the *PID* and *FOPID* controllers. The step time responses for the uncontrolled, un-tuned *PID*, tuned *PID*, un-tuned *FOPID*, and tuned *FOPID* are given in Fig. 9. The measured gain values of the two controllers, and the associated response metrics are depicted in Table I, and Table II, respectively. The results shows the best performance of the pitch control obtained from *FOPID*. It has no time delay compared to the *PID*, less rise time, less settling time, and no steady-state error.

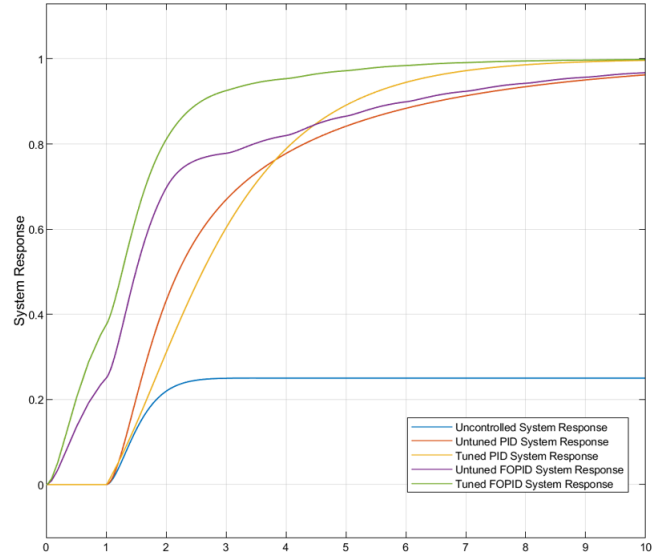


Fig. 9. Time response of the *WT* for all the scenarios.

## VI. CONCLUSION

When it comes to pitch control in wind turbine systems, *FOPID* controller has been investigated as a potential alternative to *PID* controllers. While *PID* controllers are commonly used in wind turbine applications, *FOPID* controllers

TABLE I  
GAIN VALUES OF THE CONTROLLERS.

Methodology	$K_p$	$K_i$	$K_d$	N	$\lambda$	$\mu$
Untuned <i>PID</i>	1.89	1.34	-1.156	0.56	NA	NA
Tuned <i>PID</i>	0.6	1.5	0.09	100	NA	NA
Untuned <i>FOPID</i>	1	1	0	NA	1	1
Tuned <i>FOPID</i>	0.6	1.5	0.001	NA	1.02	1.05

TABLE II  
PERFORMANCE METRICS OF ALL THE CONTROLLERS.

Controller Type	$T_r$	$T_s$	%OS	SS
untuned <i>PID</i>	4.24 sec	8.5 sec	0	0.962
tuned <i>PID</i>	3.716 sec	7.25 sec	0	0.997
untuned <i>FOPID</i>	4.76 sec	8.33 sec	0	0.968
tuned <i>FOPID</i>	2.21 sec	4.91 sec	0	1.005

offer some advantages that can lead to improved performance in certain scenarios.

Wind turbine systems often exhibit complex and non-linear behavior due to factors such as variable wind speeds, turbulence, and changes in blade aerodynamics. *FOPID* controllers, with their increased degrees of freedom (fractional order elements) can provide faster response times compared to *PID* controllers, leading to a smoother and more stable pitch control performance.

However, it's important to consider that the successful implementation of *FOPID* controllers in wind turbine pitch control requires careful tuning and parameter selection. The fractional order elements introduce additional complexity compared to *PID* controllers, and finding the optimal values for these parameters can be challenging.

*PID* and *FPID* controller technologies have ethical implications and practical applications, particularly in environmental and safety contexts. Ethical considerations involve minimizing environmental impact, prioritizing safety, and addressing algorithm biases. Practical applications include industrial processes, energy management, environmental monitoring, and autonomous systems. To ensure responsible deployment, robust system design, continuous monitoring, ethical considerations, regulatory compliance, and stakeholder engagement are crucial. Balancing operational efficiency with environmental sustainability and prioritizing safety and fairness are essential in the implementation and use of these technologies.

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